

A) Convergence to a common evaporative fluid for the SCT and Pixels

[Greg Hallewell & Vic Vacek]

and

B) Compressor Studies for fluorocarbon vapors.

[Pierre Bonneau & Vic Vacek]

Presented by

Vic Vacek

Basic properties of the candidates → fluorinerts for Atlas Pixel and SCT Cooling System					Added custom mixtures until September 99
Fluorinert	C3F8	C4F10	CF3I	Custom Mix 3_7	Custom Mix 5_5
Name	octafluoropropane	decafluorobutan	trifluoriodomethane	C3F8[30%mass] C4F10[70%mass]	C3F8[50%mass] C4F10[50%mass]
Molar mass	188.2	238.03	195.91	220.4	210.1
Triple point temperature [C]	-160.15	-84.15	-153.15	-	
Boiling point temperature [C]	-36.6	-2.09	-22	Pseudo-critical Properties:	Pseudo-critical Properties:
Critical temperature [C]	71.87	113.18	122.22	98.65	90.1
Critical pressure [bar]	26.8	23.23	38.82	25.24	25.92
Critical density [kg/m³]	628	599.8	874	608	613.6
Accentric factor [-]	0.325	0.374	0.1796	-	Custom Mix 7_3
Dipole at NBP [debye]	0.014	0	0.92	-	C3F8[70%mass] C4F10[30%mass]
Range of applicability					Molar mass = 200.7
Minimal temperature [C]	-160.2	-84.2	-93.2		T _c = 82.32
Maximal temperature [C]	226.9	226.9	146.9		P _c = 26.18
Maximal pressure [bar]	300	300	200		ρ _c = 619.3
Maximal density [kg/m ³]	2049	1823	2614		

Prediction of the thermophysical properties and their verification

- ⇒ **We are able to generate for the requested fluorinerts:**
- ⇒ Saturation tables
- ⇒ Iso-property tables
- ⇒ Single property at any state point
- ⇒ Generate appropriate diagrams
- ⇒ Predict composition of the mixtures and theirs property

Three different compositions of the C3F8/C4F10 custom mixtures were prepared and tested:

Mass fractions:

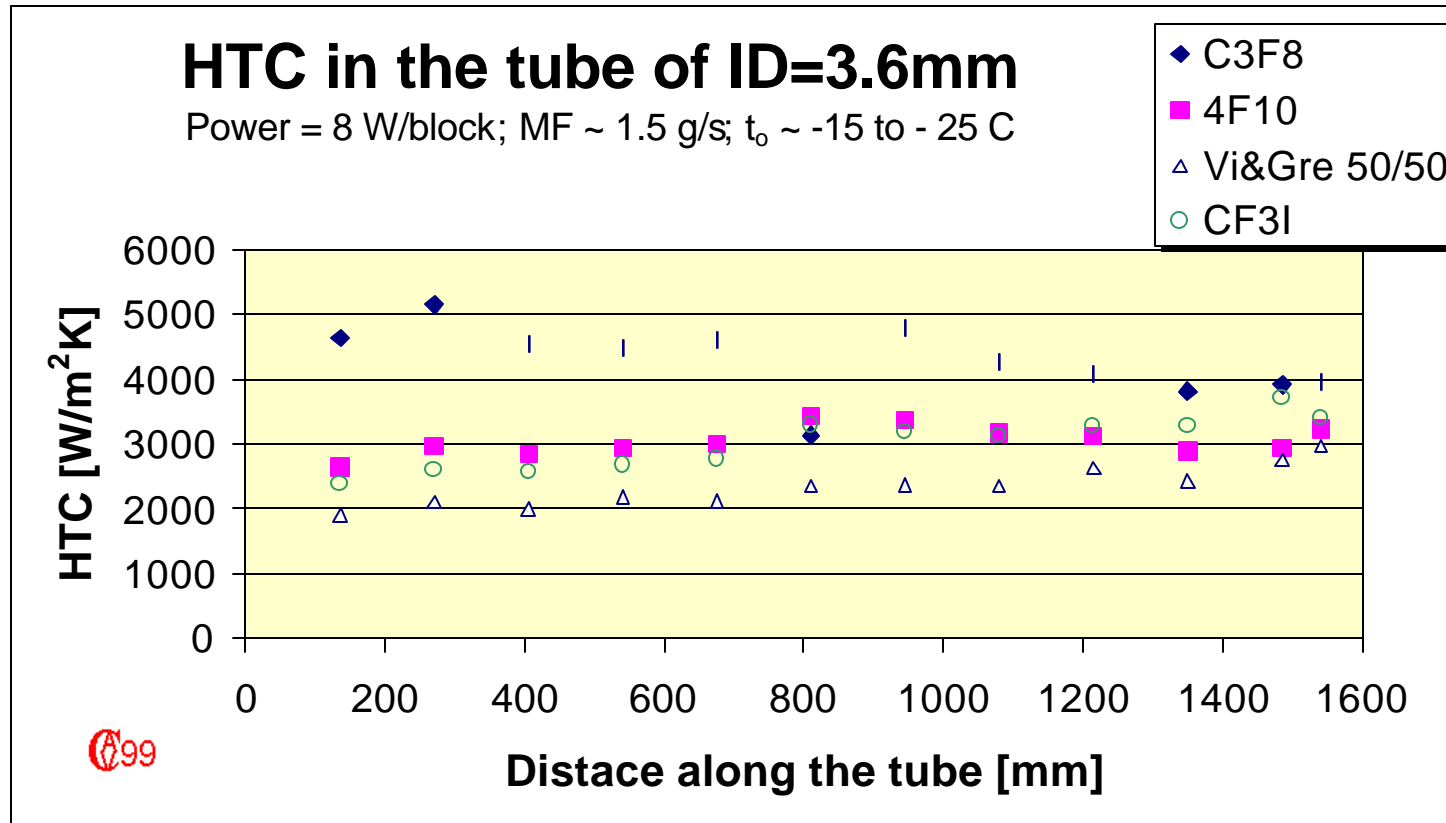
Target composition	Verified composition
(a) 0.3/0.7	[0.30/0.70]
(b) 0.5/0.5	[0.44/0.56]
(c) 0.7/0.3	[0.80/0.20]

⇒ Target composition was verified by gas chromatography analysis and via measurement of the velocity of sound using the Sonar tube developed by G. Hallewell

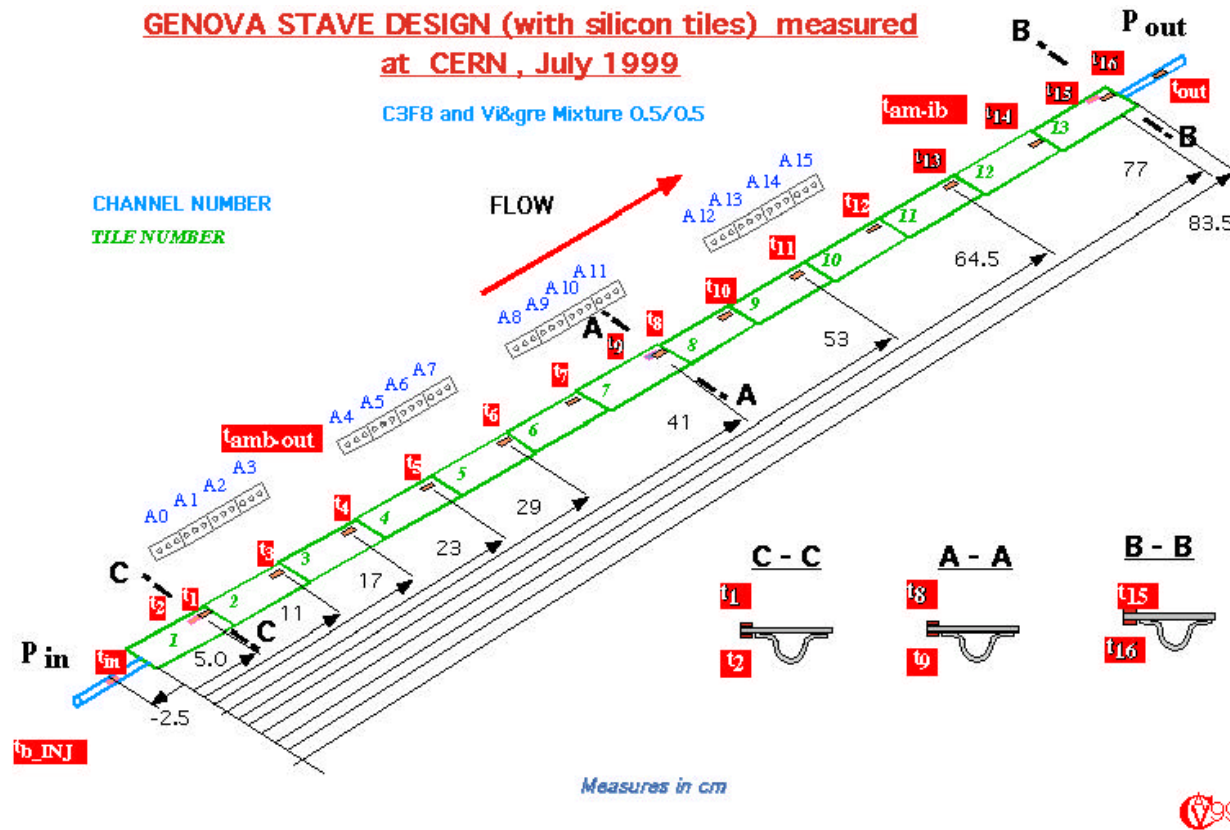
Performed measurements

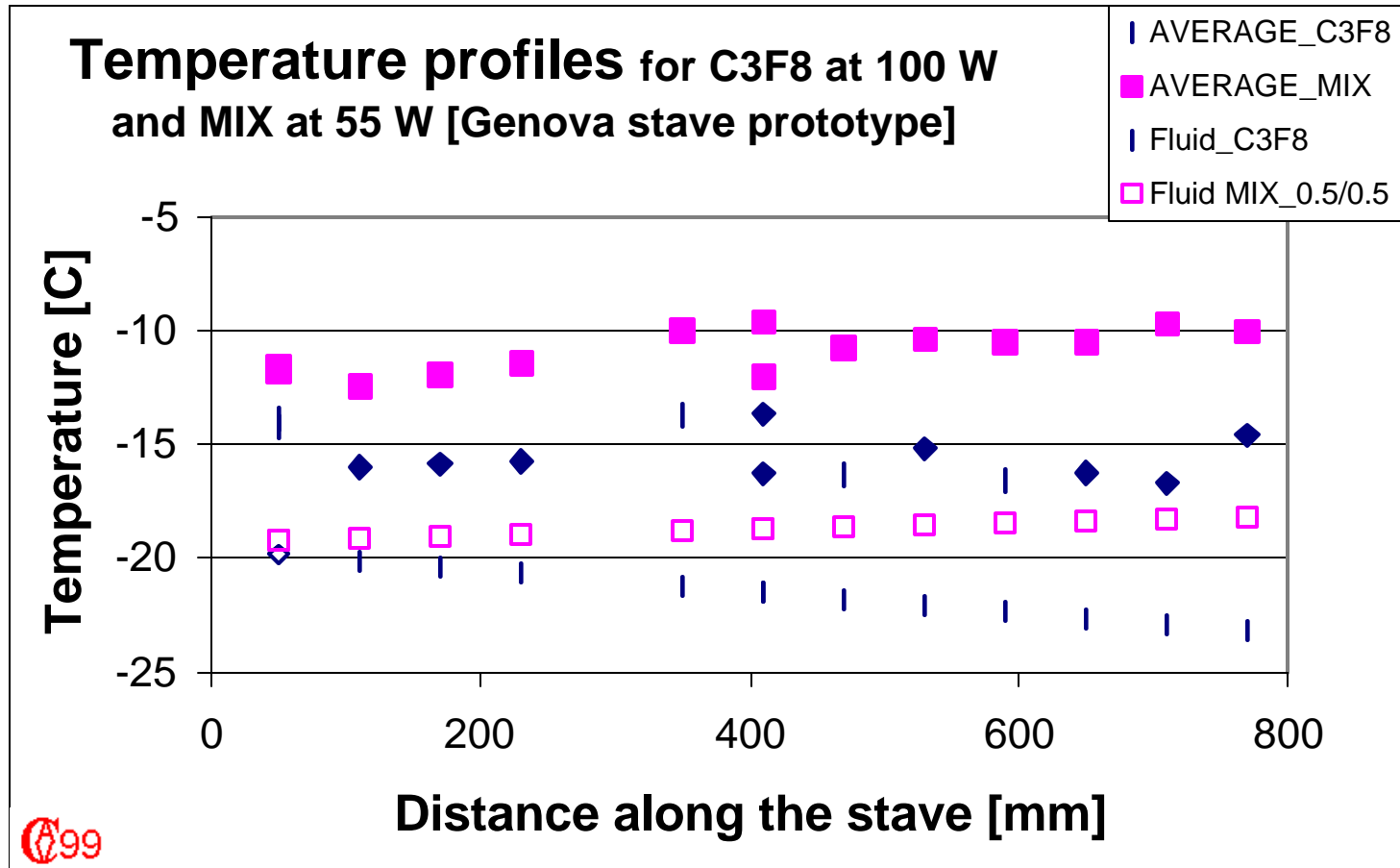
- Temperature profiles along the SCT and Pixel structures were measured
- HTC were measured for all fluids&various technological parameters
 - Different geometries [I.D. or I.D_h.]
 - Different heat fluxes
 - Different mass flows
 - Different sub-cooling
- Needle valve, ruby injectors and capillaries were tested in the evaporative circuits

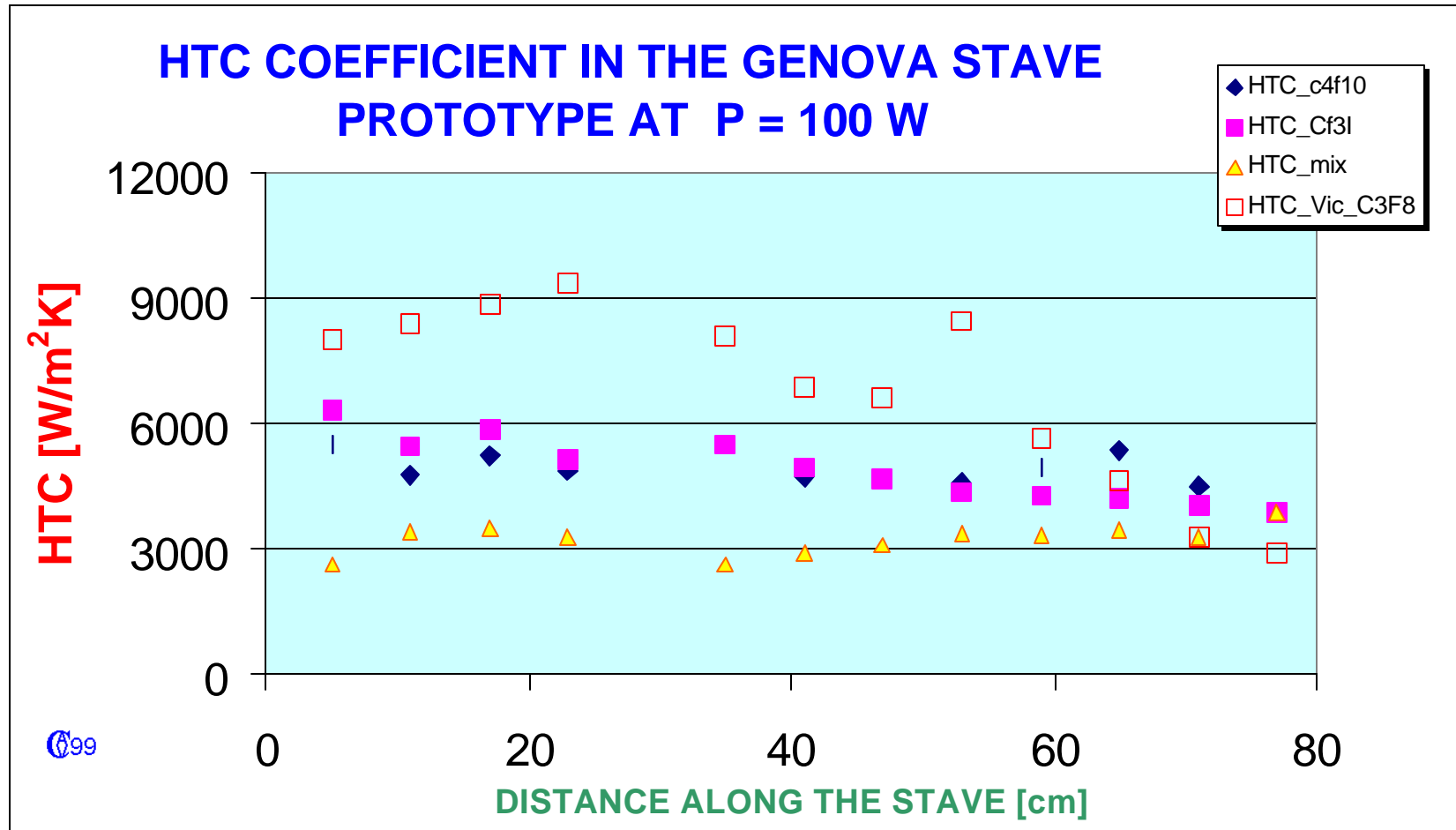
Summary results from HTC coefficient measurements

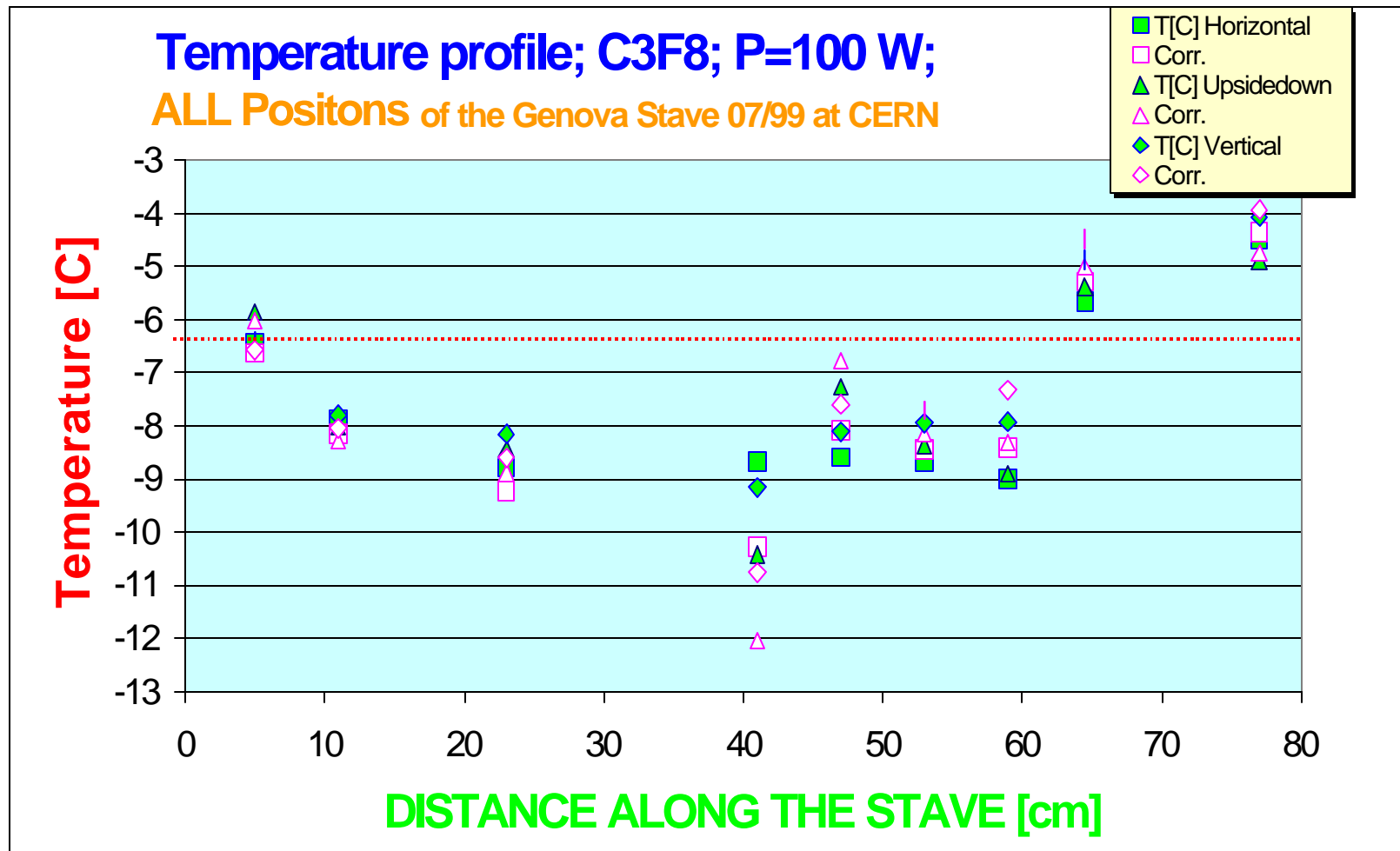


Summary results from Genova stave measurements









Conclusions

- **From the point of HTC values [averages]:**

- In the tube of ID = 3.4 mm

Tube _ I.D. =3.4 mm		HTC [W/m ² K]		
FLUID	C3F8	C4F10	MIX_ 50/50	CF3I
AVERAGE	4284	3047	2350	3024
RATIO	1.8	1.3	1.0	1.3

- In the Genova Stave Prototype

Tube _ I.D. =3.4 mm		HTC [W/m ² K]		
FLUID	C3F8	C4F10	MIX_ 50/50	CF3I
AVERAGE	6759	4881	3232	4892
RATIO	2.1	1.5	1.0	1.5

- **Other aspects are to be considered:**

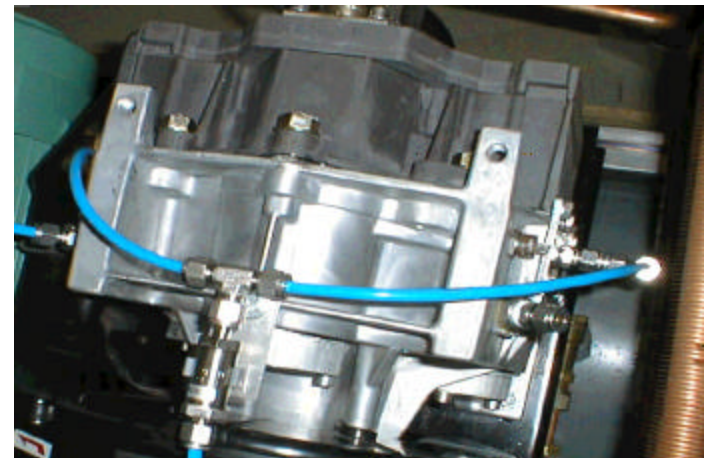
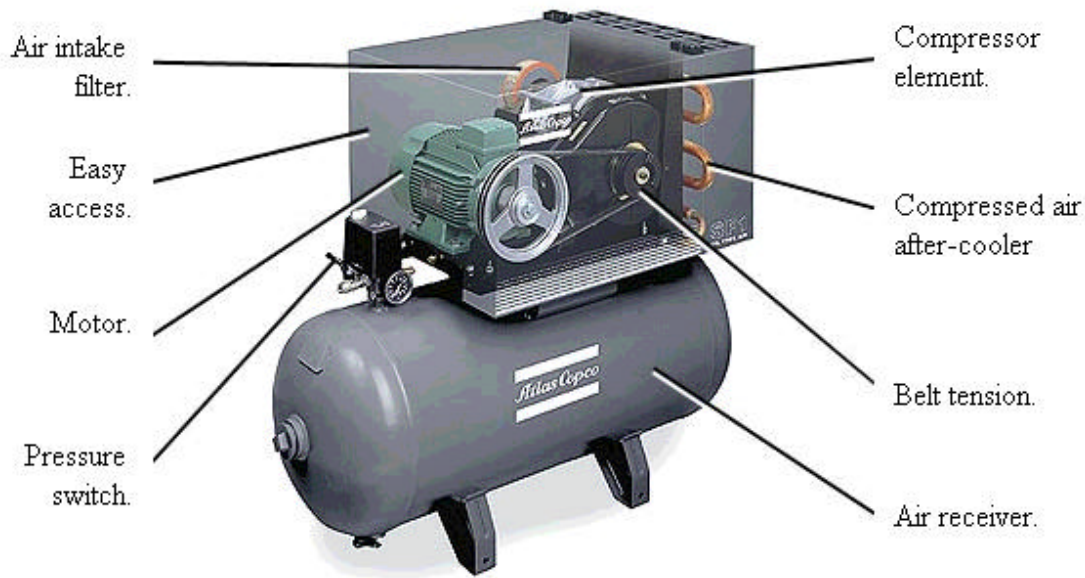
- Pressure limits for the structure
- Pressure losses within connecting pipes
- Temperature ranges [i.e. insulation matters etc.]
- Availability of the other components of the cooling circuit for certain fluid
- Compatibility with used materials
- Safety and environmental aspects
-
-
- Etc.

- HTC coefficient is not the only priority !!!

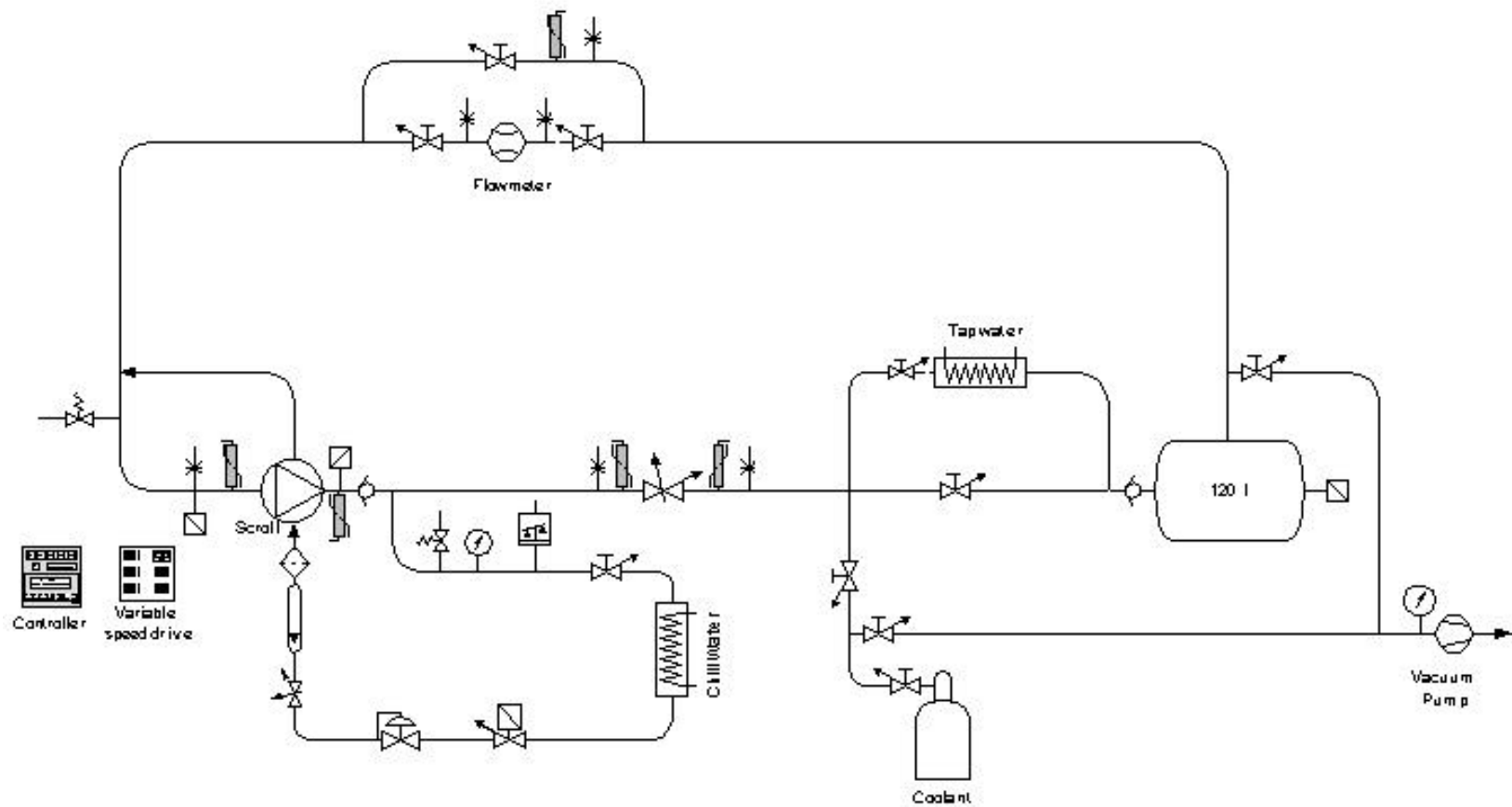
Compressor studies for fluorocarbon vapors.

- Dry scroll compressor Atlas Copco SF4-8-120

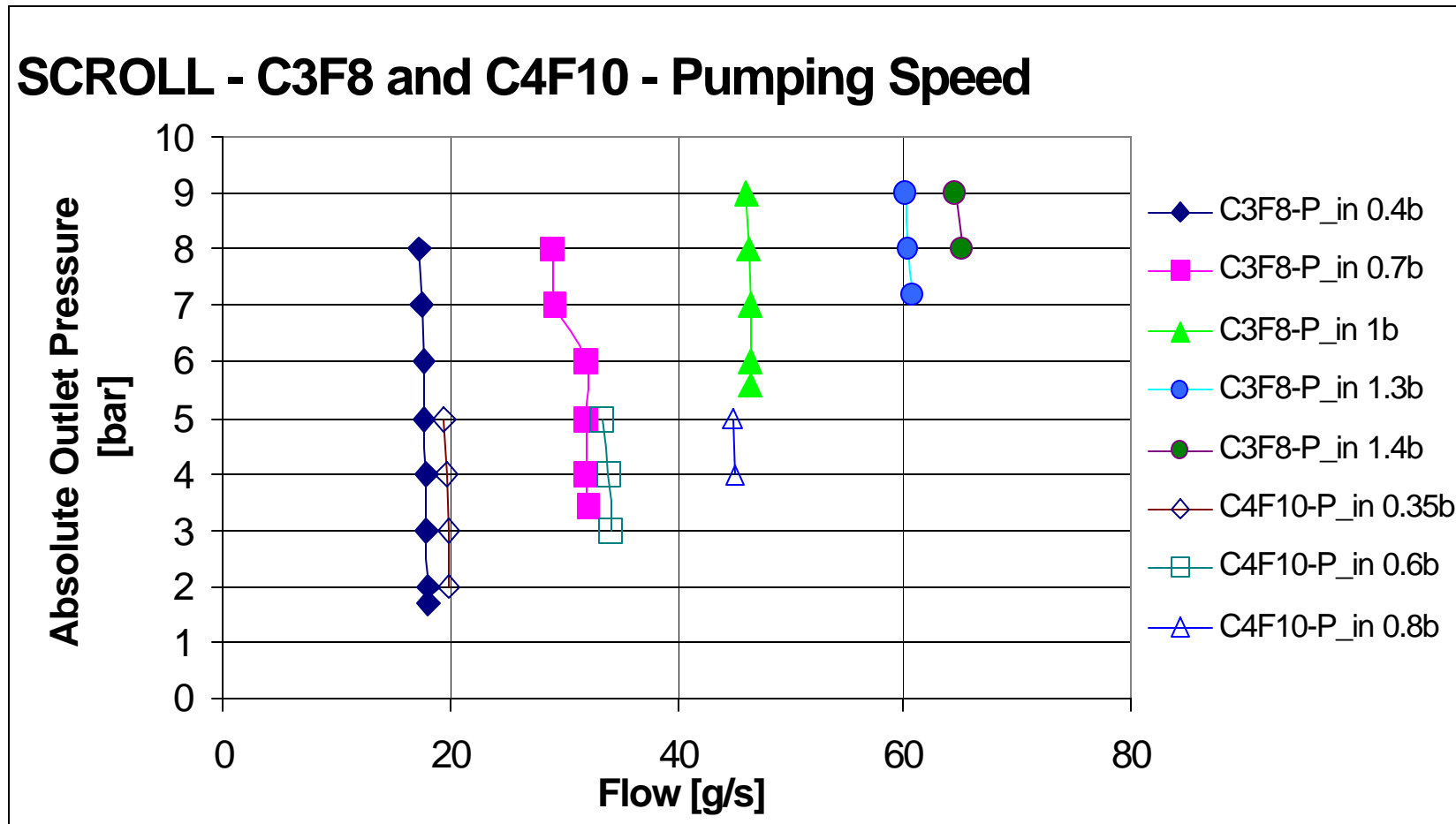
Main modifications



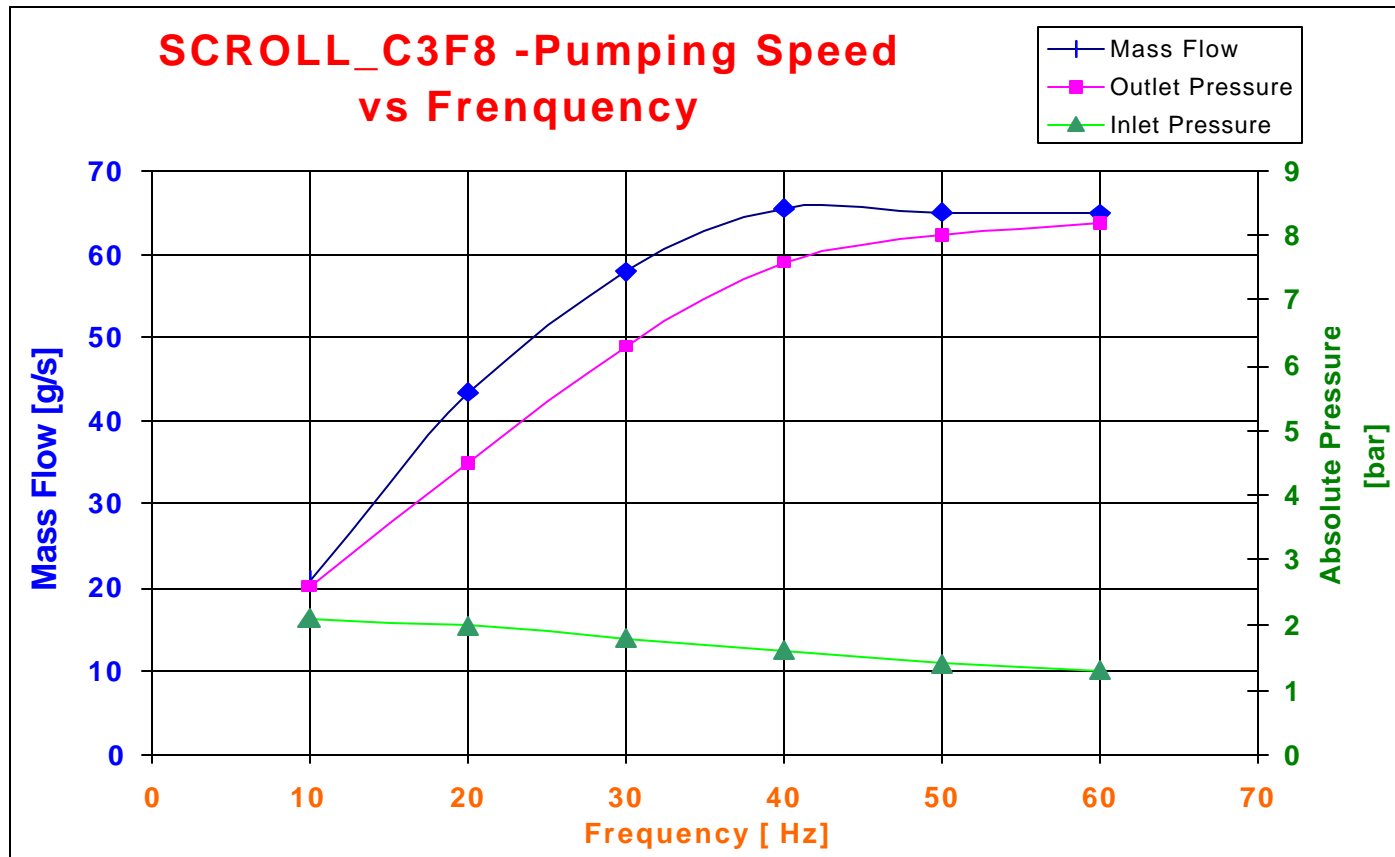
Setup for pumping speed measurement



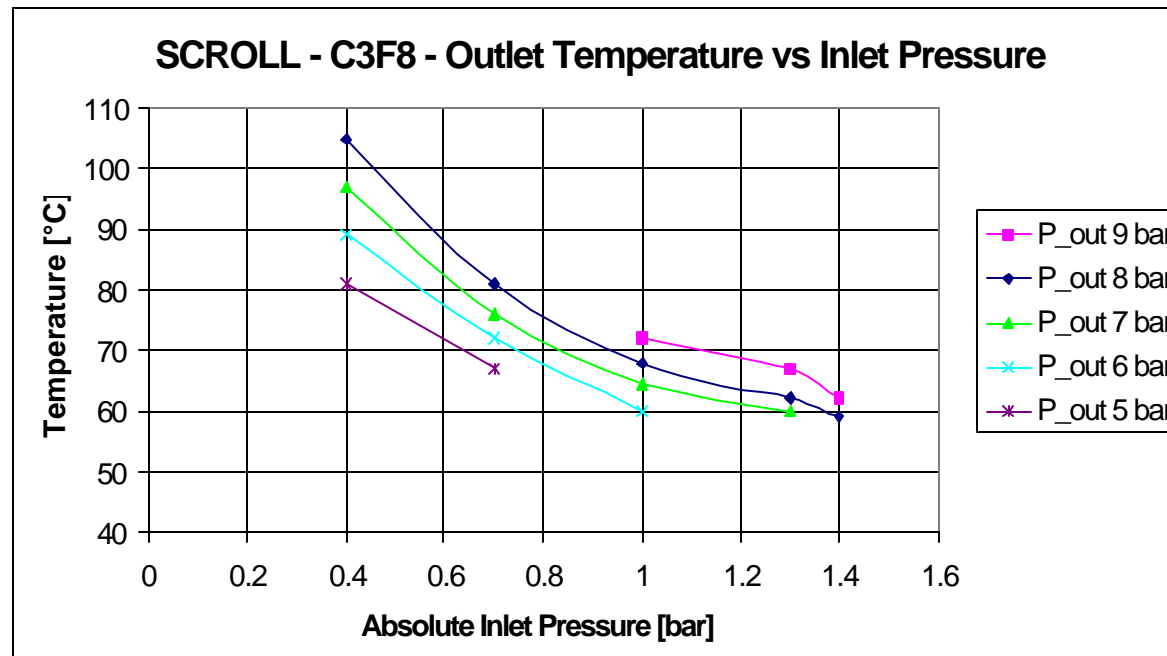
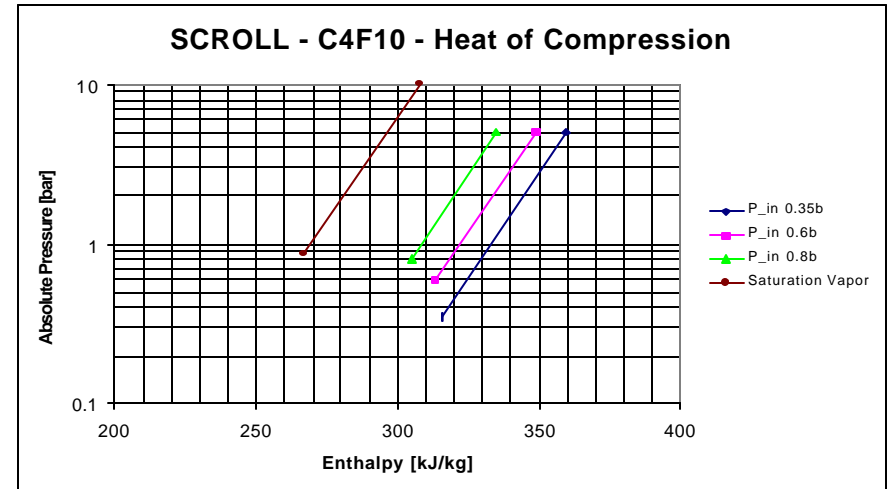
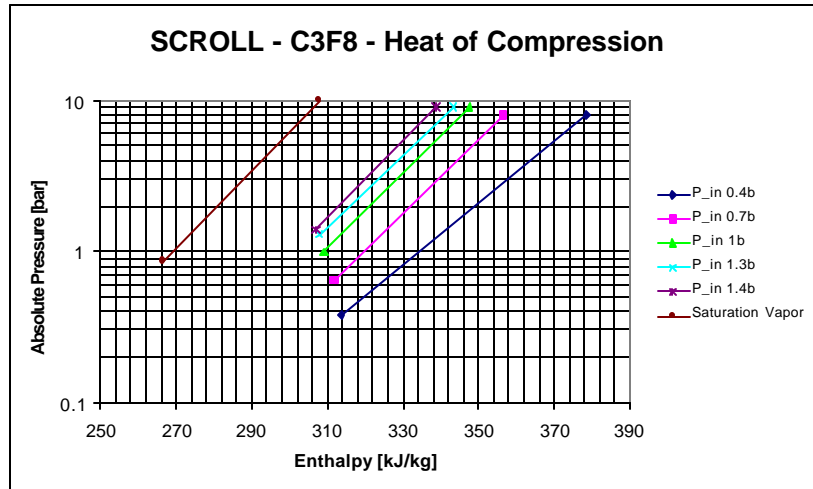
Pumping speed for the fundamental refrigerant vapors



Performance variation for C₃F₈ with frequency change



Other characteristics



Conclusions

- Successful design changes and modification include:
 - Leak-tightness of the compressor box [Helium was used for the test]
 - Internal cooling loop implementation
 - Frequency regulator device implementation
 - Buffer tank modification
- Performance test have been done with following fluids:
 - Air [initial test]
 - Fluorinert vapors:
 - C_4F_{10}
 - C_3F_8

[with an average measured flat pumping speed of $\sim 20 \text{ m}^3\text{hr}^{-1}$ for both C_4F_{10} ($P_{in} = 0.25$, $P_{out} = 4$ bar abs) and C_3F_8 ($P_{in} = 1.4$, $P_{out} = 8$ bar abs)].
- Scroll compressor is ready for an instalation into the main cooling system circuit and necessary workshop actions are under way.

